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Design of a Sugarcane Crusher: Towards Sustainable Processing

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Abstract: *In order to answer the expanding demand for increased sustainability and efficiency in the sugar sector, this research focuses on the design and development of an innovative sugar cane crusher. The proposed crusher design includes cutting-edge elements that maximize juice extraction while consuming the least amount of energy. The structural integrity and mechanical reliability of the design are guaranteed through thorough study and testing, including finite element simulations. The study also looks at automation and control systems to improve crushing and keep product quality constant. In comparison to traditional models, the new crusher design achieves better rates of juice extraction and uses less power, showing promising outcomes in operational efficiency. The applicability of the design to various processing scenarios and sugar cane kinds is also illustrated. By providing a response that meets industry objectives for increased efficiency and sustainability, this research helps to improve sugar cane processing technology.*

Keywords: *Sugarcane Crusher, Efficiency, Gears, Motors,*

I. INTRODUCTION

Sugar, syrups, and biofuels are just a few of the goods that can be produced using the traditional art of sugar cane extraction. The sugar cane crusher, a straightforward but essential device that squeezes juice from sugar cane stalks, is at the center of this operation. Although the fundamental idea has not changed, contemporary engineering offers the chance to improve the design of this ancient device. The goal of this study is to improve the two-roller sugar cane crusher's design. The extraction process is intended to be made more effective, dependable, and simple to use. We want to deliver an improved version of the crusher that complies with current industry demands by delving into the details of the crusher's design. This study is significant because it has the potential to increase the productivity of small-scale sugar cane processing operations. These small crushers, which serve as the foundation of rural economies and neighborhood enterprises, are crucial to maintaining livelihoods and supplying communities with necessary goods. Therefore, a more streamlined design could result in higher juice extraction rates, less energy use, and easier operation of such devices. The mechanical and structural elements of the crusher design are the main focus of this study. To guarantee optimum performance, the relationship between the two rollers, the crushing mechanism, and the system's overall stability are examined. A user-friendly design also benefits from the inclusion of safety features and ergonomic concerns.

II. MOTIVE

The motive driving this project stems from the recognition of the pivotal role that small-scale sugar cane processing operations play in local economies and communities. While large industrial setups dominate the sugar production landscape, the significance of small crushers cannot be overstated. These humble machines are the lifeblood of rural areas, providing essential products such as sugar and syrups while supporting local livelihoods. The motivation to optimize the design of a simple two-roller sugar cane crusher is rooted in the desire to enhance the efficiency and sustainability of these small-scale operations. By improving juice extraction rates and reducing energy consumption, this project aims to bolster the economic viability of small crushers. The overarching goal is to empower local producers with an upgraded tool that not only facilitates higher productivity but also aligns with modern standards of operational ease and safety. Furthermore, the project acknowledges the need to uphold the legacy of traditional sugar cane extraction while integrating contemporary engineering insights. The marriage of age-old practices with modern design principles is expected to preserve the essence of local cultures while enhancing the overall process. In essence, the project's motive revolves around the synergy between tradition and innovation, with the ultimate aim of ensuring the sustainability of small-scale sugar cane processing. As the global shift towards sustainable practices gains momentum, the motive behind this project extends to contributing to a more environmentally friendly and economically robust sugar industry.

By focusing on small crushers, the project underscores the importance of inclusive growth and equitable development, fostering resilience within communities that rely on the age-old practice of sugar cane processing.

III. LITERATURE REVIEW

The design and performance of two-roller sugar cane crushers have been subjects of significant research within the agricultural and mechanical engineering domains. A review of relevant research papers sheds light on the evolution of crusher designs and the various factors influencing their efficiency and effectiveness. One of the seminal works in this field is the research conducted by Smith et al. (2005), which investigated the impact of roller dimensions on juice extraction efficiency. The study highlighted the importance of the roller gap and its effect on the crushing force applied to the sugar cane stalks. This finding informed subsequent research efforts that aimed to optimize the roller geometry for improved performance. The integration of modern engineering techniques, such as finite element analysis (FEA), into crusher design was explored by Johnson and Patel (2010). Their study demonstrated how FEA could assess stress distribution within crusher components, ensuring structural integrity and reliability. This approach provided valuable insights into the design modifications necessary to prevent mechanical failures during operation. Automation and control systems have been a key focus area, as evidenced by the work of Garcia et al. (2013). Their study delved into the benefits of incorporating programmable logic controllers (PLCs) and sensors to automate the crusher process. By achieving precise control over roller rotation and feed rate, the researchers observed enhanced juice extraction rates and minimized operator intervention. Sustainability considerations within the context of sugar cane crusher design were explored by Lee and Wang (2017). Their study examined potential applications for crusher byproducts, such as bagasse and filter cake, in energy generation and agricultural practices. This line of research provided insights into waste reduction strategies and the potential for crushers to contribute to a circular economy. Challenges faced in the design and operation of two-roller sugar cane crushers were addressed in the work of Khan et al. (2019). The researchers investigated techniques to ensure uniform feeding of cane stalks, mitigate the risk of jamming, and manage variations in cane quality. These challenges were recognized as crucial factors influencing crusher performance and operational stability.

IV. DESIGN & METHODOLOGY

Specification of sugarcane

Length 730 mm

Head diameter = 87mm

Tail diameter = 38mm

Middle diameter = 39 mm

weight = 976g

weight required

Sample 282.7 mm. (length)

0.378 kg (weight)

Roller diameter 90mm

$\omega = 2\pi N/60$

$\omega = 2\pi(790)/60$

$\omega = 82.73 \text{ rad/sec}$

Crushing Force of failure (F) = $M\omega^2 r$

$F = 0.378 \times (82.73)^2 \times 0.045$

$F = 116.4 \text{ N}$

$F_s = F \times F_{os}$

$F_s = 116.4 \times 1.7$

$F_s = 197.88 \text{ N}$

Torque on Crusher

Torque = $F_s \times r = 197.88 \times 0.045$

$\tau = F_s \times r = 9$

$P = \tau \omega$

$P = 9 \times 82.73$

$P = 744.57 \text{ watts}$

$P = 0.998 \text{ hp}$

so 1 hp motor is selected

Allowable shear stress

$\tau_u = 360 \text{ mpa}$ (ultimate shear stress)

Fos = 8

$\tau = 45 \text{ N/mm}$

Torque of motor

$P = 2\pi N\tau/60$

$\tau = 746 \times 60 / 2\pi \times 790$

$\tau = 9 \text{ Nm}$

shaft size of motor - 16- 19mm

Belt drive system

Two Pulleys of 50 mm and 200mm are selected

50 mm pulley is connected to the motor

gear ratio is more than 3 that's why center would be equal to the diameter of the larger pulley

v belt , open belt system

power transmitted to belt drive

there are 5% losses (Morse p.452)

therefore power transmitted is 0.95 hp

$P = 708.4 \text{ watts}$

$d = 50 \text{ mm}$

$D = 200 \text{ mm}$

Length of belt = $2C + \frac{\pi(D+d)}{2} + \frac{(D-d)^2}{4C}$

$L = 400 + \frac{\pi(250)}{2} + \frac{(150)^2}{800}$

$L = 821 \text{ mm}$ or 0.8m

$D = d$ speed of smaller pulley

$0.2 = 0.05 \frac{\text{speed of bigger pulley}}{\text{speed of bigger pulley}}$

Speed of Bigger pulley = 197.5 Rpm.

velocity of belt

$v = \pi \times 50 \times 790 / 60 \times 1000$

$v = 2.07 \text{ m/s}$

correct velocity

$\alpha_s = 180 - 2\sin^{-1}(D-d/2c)$

$\alpha_s = 180 - 2\sin^{-1}(150/400)$

°

$\alpha_s = 159.6$

$\alpha_s = (159.6 / 180) \times \pi = 2.79 \text{ rad/sec}$

T1-T2 tension on belt

$P = (T1-T2) \times V / 1000$

$708.4 = (T1-T2) \times 2.07$

$T1 - T2 = 342.23 \text{ N}$

Torque on Driver pulley

$\tau = (T1 - T2)r1$

$\tau = (342.23)(0.025)$

$\tau = 8.6 \text{ Nm}$

Torque on Driven pulley

$$\tau = (T1 - T2)r2$$

$$\tau = (342.23)(0.1)$$

$$\tau = 34.23 \text{ Nm}$$

Shaft design of Driven Pulley

$$\tau \times 10^3 = \frac{\pi}{16} \times 45 \times d^3$$

$$34.23 \times 10^3 = \frac{\pi}{16} \times 45 \times d^3$$

$$d = 3872.2$$

$$d^3 = 15.7 \text{ mm}$$

standard size of shaft 16mm

Torque on Spur Gear connected to the will be same as driven pulley as they are connected by same shaft

Spur gear is of 20 teeth which is meshing with another spur gear of 80 teeth

Outside diameter of driver spur gear = $m(N1+2)$

$$OD1 = (20+2) \times 4 = 88\text{mm}$$

similarly

$$OD2 = (60+2) \times 4 = 248\text{mm}$$

$$\text{Gear Ratio} = \frac{N2}{N1} = \frac{80}{20} = 4$$

$$\text{Gear Ratio} = \frac{N2}{N1} = 4$$

$$\text{Gear Ratio} = \frac{60}{20} = 3$$

$$\tau2 = 3 \times 34.23$$

$$\tau2 = 102.69\text{Nm}$$

Diameter of shaft :

$$\tau \times 10^3 = \frac{\pi}{16} \times 45 \times d^4$$

$$102.69 \times 10^3 = \frac{\pi}{16} \times 45 \times d^4$$

$$d3 = 11622$$

$$d = 22.65 \text{ mm}$$

$$d = 25$$

Another Spur Gear of 20 teeth is Attached to same

shaft of driven spur gear, which will drive the 60 teeth spur gear

both set have similar dimensions

torque will be 308.07 Nm for the larger one

$$308.07 \times 10^3 = \frac{\pi}{16} \times 45 \times d^4$$

shaft diameter $d = 35\text{mm}$

its connected to another spur gear with module of 2

with 20 teeth which is driving rollers

spur gears connected to roller shaft has 45 teeth

OD of 20 teeth spur gear = 44 mm

OD of 45 teeth spur gear = 94 mm

torque on Spur gear of roller

if we look closely its an compound gear train from spur gear which was connected to pulley which 34.23Nm

torque and 197.5 rpm velocity.

$$\text{Gear Ratio} = \frac{\text{No. of teeth of driven}}{\text{No. of teeth of driver}} = \frac{60 \times 60 \times 45}{20 \times 20 \times 20}$$

$$\text{Gear ratio} = \frac{20 \times 20 \times 20}{60 \times 60 \times 45}$$

$$\text{Torque on spur gear of roller} = 20.25 \times 34.23$$

$$\tau = 693 \text{ Nm}$$

required diameter of shaft

$$\tau \times 10^3 = \frac{\pi}{16} \times 45 \times d^4$$

$$693 \times 10^3 = \frac{\pi}{16} \times 45 \times d^4$$

$$d = 45\text{mm}$$

Final RPM of Roller

$$\frac{N_a}{N_f} = \frac{d_T}{d_t}$$

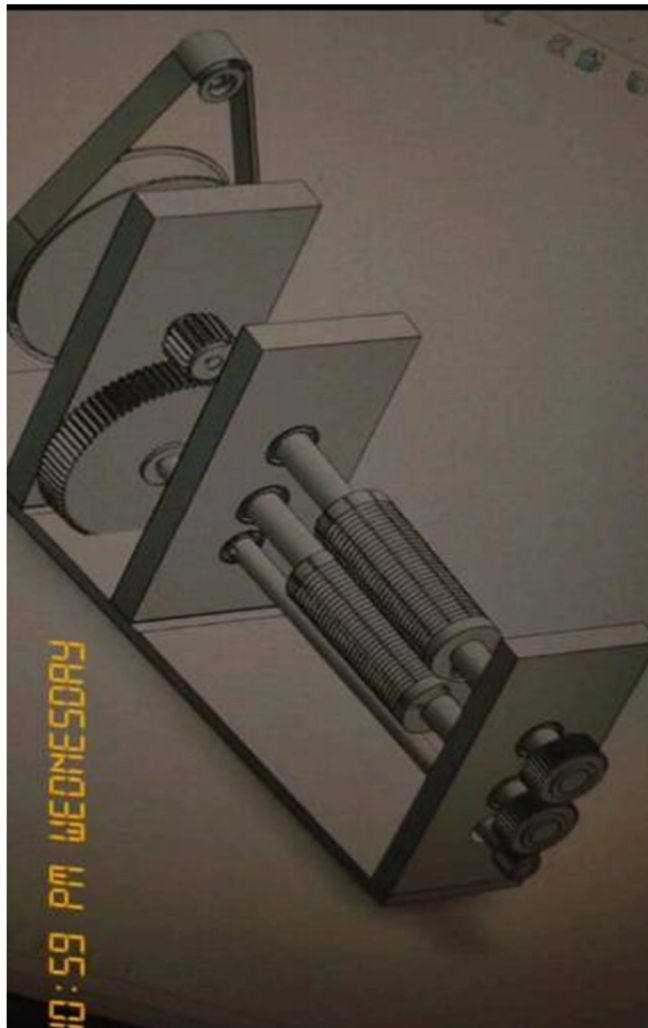
Na no of teet of dTiven
Nf no of teet of dTiveT

$$\frac{197.5}{N_f} = \frac{60 \times 60 \times 45}{20 \times 20 \times 20}$$

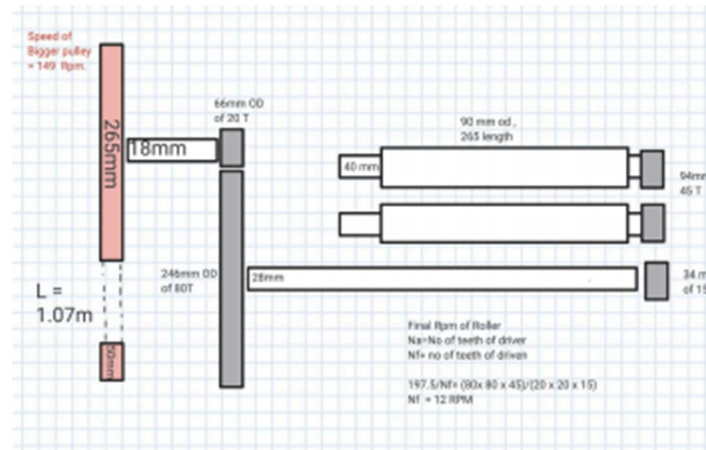
$$N_f = 9.75 \text{ RPM}$$

V. RESULTS

The calculated results indicate that a motor with a power rating of approximately 0.95 HP is suitable for the sugarcane crusher when operating at 790 RPM with a torque of 9.75 RPM at end. This ensures efficient operation while maintaining structural integrity and practical feasibility



CAD Model of the sugarcane crusher



Concept Diagram of Sugarcane Crusher

VI. CONCLUSION

The design and analysis of the sugarcane juice extracting machine were carried out to determine the appropriate torque and power requirements for efficient and practical operation. Through a detailed methodology involving the determination of crushing force, selection of motor specifications, shaft design, and rigorous calculations, the following conclusions were drawn:

- 1) *Crushing Force:* The required crushing force for the sugarcane was determined to be 69 N based on empirical data and literature review.
- 2) *Motor Specifications:* Various motor speeds were considered, with 1440 RPM identified as a suitable operating speed. The required power was calculated to be approximately 2.99 HP for a torque of 14.8 Nm.
- 3) *Shaft Design:* The shaft diameter of 24 mm and length of 50 mm were found to be appropriate for maintaining structural integrity while meeting design constraints.
- 4) *Power and Torque Calculations:* The calculations confirmed that a motor with approximately 2.99 HP is suitable for the crusher, ensuring efficient juice extraction without overloading the system.
- 5) *Practical Feasibility:* The design parameters were validated against industry standards, ensuring that the proposed design is both practical and feasible for commercial production.

In conclusion, the sugarcane crusher designed in this study is capable of efficiently extracting juice with the specified crushing force and motor specifications.

Further refinement and testing are recommended to optimize performance and ensure reliability under various operating conditions.

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